## BULLETIN

DE

# L'ACADÉMIE POLONAISE DES SCIENCES

Rédacteur en chef K. KURATOWSKI Rédacteur en chef suppléant L INFELD

SÉRIE DES SCIENCES TECHNIQUES

Rédacteur de la Série J. GROSZKOWSKI

Comité de Rédaction de la Série
C. KANAFOJSKI, W. NOWACKI, W. OLSZAK, B. STEFANOWSKI,
P. SZULKIN, W. SZYMANOWSKI

VOLUME VII NUMÉRO 1

APPLIED MECHANICS

### Thermal Stresses in Orthotropic Plates

by

#### W. NOWACKI

Presented on September 8, 1958

In many cases of engineering practice we meet orthogonally anisotropic ("orthotropic") plates, showing different elastic and thermal properties in two orthogonal directions. By E, and E, we denote Young's moduli in the direction of the  $x_1$  and  $x_2$  axis, respectively, by  $v = v_{12}$ Poisson's ratio and by  $G = G_{12}$  the shear modulus. Finally  $a_1$  and  $a_2$ denote the coefficients of thermal expansion and  $\lambda_1$ ,  $\lambda_2$  coefficients of thermal conductivity in the direction of the  $x_1$  and  $x_2$  axes, respectively The heat equation for an orthotropic plate has the form

(1) 
$$\lambda_1 \frac{\partial^2 T}{\partial x_1^2} + \lambda_2 \frac{\partial^2 T}{\partial x_2^2} - c\varrho \frac{\partial T}{\partial t} = -W,$$

where c is the specific heat  $\rho$  — the density and W — the rate of heat generated pro unity of volume and time. The relations between stress and strain in the plane state of stress are [1],

(2) 
$$\begin{cases} \varepsilon_{11} = a_{11} \, \sigma_{11} + a_{12} \, \sigma_{12} + a_{1} \, T, \\ \varepsilon_{22} = a_{21} \, \sigma_{11} + a_{22} \, \sigma_{22} + a_{2} \, T, \\ \varepsilon_{12} = a_{66} \, \sigma_{12}, \quad a_{12} = a_{21}, \end{cases}$$

where

$$a_{11} = \frac{1}{E_1}$$
,  $a_{22} = \frac{1}{E_2}$ ,  $a_{12} = a_{21} = -\frac{r_1}{E_1}$ ,  $a_{66} = \frac{1}{2G}$ .

Substituting the strains in the compatibility equation

(3) 
$$\frac{\partial^2 \varepsilon_{11}}{\partial x_2^2} + \frac{\partial^2 \varepsilon_{22}}{\partial x_1^2} = 2 \frac{\partial^2 \varepsilon_{12}}{\partial x_1 \partial x_2},$$

we have

(4) 
$$\left( a_{11} \frac{\partial^2}{\partial x_2^2} + a_{12} \frac{\partial^2}{\partial x_1^2} \right) \sigma_{11} + \left( a_{12} \frac{\partial^2}{\partial x_2^2} + a_{22} \frac{\partial^2}{\partial x_1^2} \right) \sigma_{22} - \frac{\partial^2}{\partial x_1^2} - 2 a_{00} \frac{\partial^2}{\partial x_1} \frac{\sigma_{12}}{\partial x_2} + \left( a_1 \frac{\partial^2}{\partial x_2^2} + a_2 \frac{\partial^2}{\partial x_1^2} \right) T = 0.$$

Let us express the stresses by means of the Airy function

(5) 
$$\sigma_{11} = \frac{\partial^2 F}{\partial x_2^2}, \quad \sigma_{22} = \frac{\partial^2 F}{\partial x_1^2}, \quad \sigma_{12} = -\frac{\partial^2 F}{\partial x_1 \partial x_2},$$

and substitute them in (4). After some simple transformations we obtain the differential equation

(6) 
$$\frac{\partial^4 F}{\partial x_1^4} \varkappa^4 + 2 \eta \varkappa^2 \frac{\partial^4 F}{\partial x_1^2 \partial x_2^2} + \frac{\partial^4 F}{\partial x_2^4} + E_1 \left( a_1 \frac{\partial^2}{\partial x_2^2} + a_2 \frac{\partial^2}{\partial x_1^2} \right) T = 0,$$

where

$$\kappa^4 = \frac{E_1}{E_2}, \quad 2 \eta \kappa^2 = E_1 \left( \frac{1}{G} - \frac{2 \nu_1}{E_1} \right).$$

Let us compose the solution of Eq. (6) of two components  $\Phi$  and  $\Psi$ , where the function  $\Psi$  is a particular integral of Eq. (6). It therefore satisfies equation

(7) 
$$\frac{\partial^4 \Psi}{\partial x_1^4} \kappa^4 + 2 \eta \kappa^2 \frac{\partial^4 \Psi}{\partial x_1^2 \partial x_2^2} + \frac{\partial^4 \Psi}{\partial x_2^4} + E_1 \left( \alpha_1 \frac{\partial^2}{\partial x_2^2} + \alpha_2 \frac{\partial^2}{\partial x_1^2} \right) T = 0,$$

the function  $\Phi$  satisfying the quasi-biharmonic homogeneous equation

(8) 
$$\frac{\partial^4 \Phi}{\partial x_1^4} \varkappa^4 + 2 \eta \varkappa^2 \frac{\partial^4 \Phi}{\partial x_1^2 \partial x_2^2} + \frac{\partial^4 \Phi}{\partial x_2^4} = 0$$

and the boundary conditions.

The resulting stresses  $\sigma_{ij}$  will be obtained from the equations

$$\sigma_{ij} = \overline{\sigma}_{ij} + \overline{\sigma}_{ij} = \left( \nabla^2 \delta_{ij} - \frac{\partial^2}{\partial x_i \partial x_j} \right) (\Psi + \Phi) \quad i, j = 1, 2.$$

The procedure just described is particularly convenient in the case of boundary conditions expressed in stresses.

If the boundary conditions are given in displacements, the following method is preferable.

We solve the system of Eqs. (2) for stresses

(9) 
$$\begin{cases} \sigma_{11} = A_{11} \, \varepsilon_{11} + A_{12} \, \varepsilon_{22} - \beta_1 \, T, \\ \sigma_{22} = A_{21} \, \varepsilon_{11} + A_{22} \, \varepsilon_{22} - \beta_2 \, T, \\ \sigma_{12} = 2 \, A_{66} \, \varepsilon_{12}, \end{cases}$$

where

$$\begin{split} A_{11} &= \frac{E_1^2}{E_1 - v_1^2 E_2}, \quad A_{22} = \frac{E_1 E_2}{E_1 - v_1^2 E_2}, \quad A_{12} = \frac{E_1 E_2 v_1}{E_1 - v_1^2 E_2}, \quad A_{66} = G, \\ \beta_1 &= \frac{E_1^2 (a_1 + a_2 v_2)}{E_1 - v_1^2 E_2}, \quad \beta_2 = \frac{E_1 E_2 (a_2 + a_1 v_1)}{E_1 - v_1^2 E_2}, \quad E_1 v_2 = E_2 v_1. \end{split}$$

Then, we substitute (9) in the equilibrium conditions

(10) 
$$\sum_{j} \frac{\partial \sigma_{ij}}{\partial x_{j}} = 0 \quad i, j = 1, 2$$

and express the strains in terms of displacements

(11) 
$$2 \varepsilon_{ij} = \frac{\partial u_i}{\partial x_i} + \frac{\partial u_j}{\partial x_i} \qquad i, j = 1, 2.$$

Thus, we obtain the system of equations

Let us join to this Eq. (1)

(13) 
$$\lambda_1 \frac{\partial^2 T}{\partial x_1^2} + \lambda_2 \frac{\partial^2 T}{\partial x_2^2} - c\varrho \frac{\partial T}{\partial t} = -W.$$

The system of Eqs. (12), (13) may be expressed in the operational form

(14) 
$$\sum_{i=1}^{j=3} L_{ij} u_j = -W \delta_{3i} \quad i=1,2,3,$$

where

$$egin{align*} L_{11} &= A_{11} rac{\partial^2}{\partial x_1^2} + A_{66} rac{\partial^2}{\partial x_2^2}, & L_{22} &= A_{66} rac{\partial^2}{\partial x_1^2} + A_{22} rac{\partial^2}{\partial x_2^2}, \ & L_{12} &= L_{21} = (A_{12} + A_{66}) rac{\partial^2}{\partial x_1 \partial x_2}, & L_{13} &= -eta rac{\partial}{\partial x_1}, & L_{23} &= -eta rac{\partial}{\partial x_2}, \ & L_{33} &= \lambda_1 rac{\partial^2}{\partial x_1^2} + \lambda_2 rac{\partial^2}{\partial x_2^2} - c arrho rac{\partial}{\partial t}, & L_{31} &= 0, & L_{32} &= 0 \end{split}$$

it being assumed that  $u_3 = T$ .

The functions  $u_i$  (i = 1, 2, 3) may be expressed by means of the three functions  $\chi_i$  (i = 1, 2, 3), as follows:

$$(15) \quad u_{1} = \begin{vmatrix} \chi_{1}, & L_{12}, & L_{13} \\ \chi_{2}, & L_{22}, & L_{23} \\ \chi_{3}, & 0, & L_{33} \end{vmatrix} \quad u_{2} = \begin{vmatrix} L_{11}, & \chi_{1}, & L_{13} \\ L_{21}, & \chi_{2}, & L_{23} \\ 0, & \chi_{3}, & L_{33} \end{vmatrix} \quad u_{3} = \begin{vmatrix} L_{11}, & L_{12}, & \chi_{1} \\ L_{21}, & L_{22}, & \chi_{2} \\ 0, & 0, & \chi_{3} \end{vmatrix},$$

or, after performing the operations prescribed,

$$\begin{cases} u_{1} = L_{33} \left( A_{66} \frac{\partial^{2}}{\partial x_{1}^{2}} + A_{22} \frac{\partial^{2}}{\partial x_{2}^{2}} \right) \chi_{1} - L_{33} \left( A_{12} + A_{66} \right) \frac{\partial^{2}}{\partial x_{1}} \chi_{2} + \\ + \beta_{1} \frac{\partial}{\partial x_{1}} \left( A_{66} \frac{\partial^{2}}{\partial x_{1}^{2}} + \overline{A}_{22} \frac{\partial^{2}}{\partial x_{2}^{2}} \right) \chi_{3} , \\ u_{2} = -L_{33} \left( A_{12} + A_{66} \right) \frac{\partial^{2}}{\partial x_{1}} \chi_{1} + L_{33} \left( A_{11} \frac{\partial^{2}}{\partial x_{1}^{2}} + A_{66} \frac{\partial^{2}}{\partial x_{2}^{2}} \right) + \\ + \beta_{2} \frac{\partial}{\partial x_{2}} \left( A_{66} \frac{\partial^{2}}{\partial x_{2}^{2}} + \overline{A}_{11} \frac{\partial^{2}}{\partial x_{1}^{2}} \right) \chi_{3} , \\ u_{3} = T = A_{22} A_{66} \left( \frac{\partial^{4}}{\partial x_{2}^{4}} + 2 \sigma \varkappa^{2} \frac{\partial^{4}}{\partial x_{1}^{2} \partial x_{2}^{2}} + \varkappa^{4} \frac{\partial^{4}}{\partial x_{1}^{4}} \right) \chi_{3} , \end{cases}$$

where

The functions  $\chi_i$  (i=1, 2, 3) satisfy the operational equation

(17) 
$$\begin{vmatrix} L_{11}, L_{12}, L_{13} \\ L_{21}, L_{21}, L_{23} \\ 0, 0, L_{33} \end{vmatrix} \chi_{i} = -W \delta_{3i} \quad i = 1, 2, 3,$$

or

$$A_{22}\,A_{66}\,L_{33}\left(\mu_1^2rac{\partial^2}{\partial x_1^2}+rac{\partial^2}{\partial x_2^2}
ight)\left(\mu_2^2rac{\partial^2}{\partial x_1^2}+rac{\partial^2}{\partial x_2^2}
ight)\chi_i\!=\!-W\delta_{3i},\quad i\!=\!1,2,3,$$

where

$$\mu_{1,2}^2 = \varkappa^2 \left\{ egin{array}{ll} \sigma \pm \sqrt{\sigma^2 - 1} & ext{for} & \sigma > 1, \\ \sigma & ext{for} & \sigma = 1, \\ \left(\sqrt{rac{1+\sigma}{2}} \pm \sqrt{rac{1-\sigma}{2}}
ight)^2 & ext{for} & 0 < \sigma < 1. \end{array} 
ight.$$

The functions  $\chi_1$ ,  $\chi_2$  satisfy the homogeneous equation and the function  $\chi_3$  — the non-homogeneous differential equation. The functions  $\chi_1$ ,  $\chi_2$  are B. G. Galerkin's functions generalized to the case of orthotropy [2].

The solution procedure is as follows. From the Eq. (17") we determine, for i=3, the particular integral  $\chi_3$ . By means of the functions  $\chi_1$ ,  $\chi_2$  we satisfy the given boundary conditions in displacements. Substituting the quantities  $u_i$  (i=1, 2) in relations (9), we obtain the stresses.

The determination of the thermal stresses is particularly simple for an infinite orthotropic plate. In this particular case it is most convenient to use Eq. (6).

Let the temperature  $T_0 = \text{const.}$  be prescribed in the region of the rectangle of sides c, d in an infinite plate. Let the T temperature outside this region be zero.

The temperature field is represented by the Fourier integral

(18) 
$$T = \frac{4T_0}{\pi^2} \int_0^{\infty} \int_0^{\infty} \frac{\sin \alpha c \sin \beta d}{\alpha \beta} \cos \alpha x_1 \cos \beta x_2 d\alpha d\beta.$$

Representing the Airy function also by means of a Fourier integral

(19) 
$$F = \frac{4}{\pi^2} \int_0^\infty \int_0^\infty A(\alpha, \beta) \cos \alpha x_1 \cos \beta x_2 \, d\alpha \, d\beta,$$

and substituting (18) and (19) into (6), we obtain  $A(\alpha, \beta)$  and the function F in the form

(20) 
$$F = \frac{4 T_0 E_1}{\pi^2} \int_0^{\infty} \int_0^{\infty} \frac{\sin \alpha c \sin \beta d}{\alpha \beta} \frac{a_1 \beta^2 + a_2 \alpha^2}{\alpha^4 \varkappa^4 + 2 \eta \varkappa^2 \alpha^2 \beta^2 + \beta^4} \cos \alpha x_1 \cos \beta x_2 d\alpha d\beta.$$

Bearing in mind that

$$\alpha^4 \varkappa^4 + 2 \eta \varkappa^2 \alpha^2 \beta^2 + \beta^4 = (\alpha^2 \gamma_1^2 + \beta^2) (\alpha^2 \gamma_2^2 + \beta^2),$$

where

$$\gamma_{1,2}^{2:} \! = \! \varkappa^2 \left\{ egin{array}{ll} \eta \! \pm \! \sqrt{\eta^2 - 1} & ext{for} & \eta \! > \! 0, \\ \eta & ext{for} & \eta \! = \! 1, \\ \left( \sqrt{rac{1 + \eta}{2}} \! \pm \! \sqrt{rac{1 - \eta}{2}} 
ight)^{\!\! 2} & ext{for} & 0 \! < \! \eta \! < \! 1, \end{array} 
ight.$$

and introducing the notations

$$\gamma_3^2 = \frac{a_2}{a_1}, \quad a_1 = \frac{\gamma_1^2 - \gamma_3^2}{\gamma_1^2 - \gamma_2^2}, \quad a_2 = \frac{\gamma_2^2 - \gamma_3^2}{\gamma_2^2 - \gamma_1^2},$$

we represent the function F in the form

(21) 
$$F = \frac{4 T_0 a_1 E_1}{\pi^2} \int_0^{\infty} \int_0^{\infty} \frac{\sin ac \sin \beta d}{a\beta} \left[ \frac{a_1}{\gamma_1^2 a^2 + \beta^2} + \frac{a_2}{\gamma_2^2 a^2 + \beta^2} \right] \cos ax_1 \cos \beta x_2 \, dad\beta.$$

The stresses  $\sigma_{ij}$  will be determined by means of Eqs. (5). Thus, for instance, for stresses  $\sigma_{i2}$  and  $\sigma_{11}$ , we obtain, after the operations indicated, the following closed expressions

$$\begin{split} \sigma_{12} &= -\frac{\partial^2 F}{\partial x_1 \partial x_2} = \\ &= -\frac{T_0 a_1 E_1}{2 \pi} \Big\{ \frac{a_1}{\gamma_1} \ln \frac{[\gamma_1^2 (x_2 + d)^2 + (x_1 - c)^2] \left[ \gamma_1^2 (x_2 - d)^2 + (x_1 + c)^2 \right]}{[\gamma_1^2 (x_2 - d)^2 + (x_1 - c)^2] \left[ \gamma_1^2 (x_2 + d)^2 + (x_1 + c)^2 \right]} + \\ &\quad + \frac{a_2}{\gamma_2} \ln \frac{[\gamma_2^2 (x_2 + d)^2 + (x_1 - c)^2] \left[ \gamma_2^2 (x_2 - d)^2 + (x_1 + c)^2 \right]}{[\gamma_2^2 (x_2 - d)^2 + (x_1 - c)^2] \left[ \gamma_2^2 (x_2 + d)^2 + (x_1 + c)^2 \right]} \Big\}, \\ \sigma_{11} &= \frac{\partial^2 F}{\partial x_2^2} = -\frac{T_0 a_1 E_1}{2 \pi} \Big\{ a_1 \left[ tg^{-1} \frac{x_1 - c}{\gamma_1 (x_2 - d)} - tg^{-1} \frac{x_1 - c}{\gamma_1 (x_2 + d)} \right] - tg^{-1} \frac{x_1 + c}{\gamma_1 (x_2 - d)} + tg^{-1} \frac{x_1 + c}{\gamma_1 (x_2 + d)} \Big\} + a_2 \left[ tg^{-1} \frac{x_1 - c}{\gamma_2 (x_2 - d)} - tg^{-1} \frac{x_1 - c}{\gamma_2 (x_2 - d)} \right] \Big\}, \end{split}$$

 $tg^{-1}z = arc tg z$ .

These expressions are valid for  $\eta > 1$ .

It is evident that, if the "corner" of the rectangle is approached, the stresses  $\sigma_{12}$ ,  $\sigma_{11}$  increase indefinitely, and the stresses  $\sigma_{11}$  show discontinuities in the cross-sections  $x_2 = \pm d$ .

DEPARTMENT OF MECHANICS OF CONTINUOUS MEDIA, INSTITUTE OF BASIC TECHNICAL PROBLEMS, POLISH ACADEMY OF SCIENCES

(ZAKŁAD MECHANIKI OŚRODKÓW CIĄGŁYCH, INSTYTUT PODSTAWOWYCH PROB-LEMÓW TECHNIKI, PAN).

#### REFERENCES

[1] S. G. Lekhnitzky, Anisotropic plates (in Russian), Ogiz., Moscow (1947).
[2] B. G. Galerkin, Determination of stresses and strains in an isotropic

[2] B. G. Galerkin, Determination of stresses and strains in an isotropic elastic body with the help of three functions (in Russian), Izw. Nauczn.-issl. Inst. Gidrotechn., 1, Leningrad (1931).